NATIONAL EXAMINATIONS DECEMBER 2018

17-Phys-B6, Applied Thermodynamics and Heat Transfer

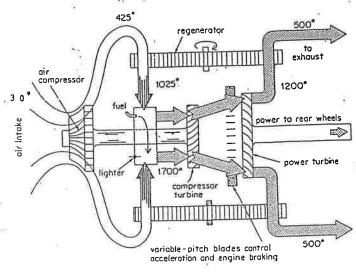
3 Hours Duration

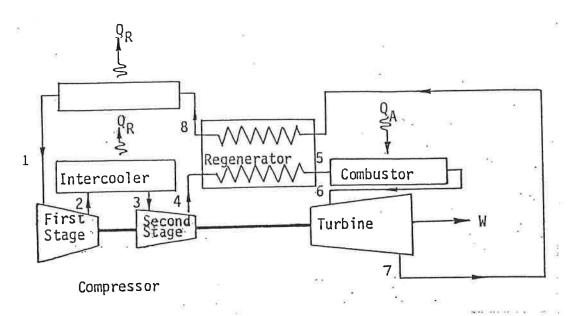
Notes:

- 1. If doubt exists concerning the interpretation of any question, the candidate is urged to make assumptions and clearly explain what has been assumed along with the answer to the question.
- 2. The examination is open book. As a consequence, candidates are permitted to make use of any textbooks, references or notes.
- 3. Any non-communicating calculator is permitted. However, candidates must indicate the type of calculator(s) that they have used by writing the name and model designation of the calculator(s) on the inside of the cover of the first examination book.
- 4. It is expected that each candidate will have copies of <u>both</u> a thermodynamics text <u>and</u> a heat transfer text in order to make use of the information presented in the tables and graphs contained.
- 5. The answers to <u>five</u> questions, <u>either</u> three questions from Part A and two questions from Part B <u>or</u> two questions from Part A and three questions from Part B, comprise a complete examination.
- 6. Candidates <u>must</u> indicate the answers that they wish to have graded on the cover of the first examination book. Otherwise the answers will be graded in the order in which they appear in the examination book(s) up to a maximum of three answers per section.
- 7. The answer to any question carries the same value in the grading.

PART A - THERMODYNAMICS

- (a) A closed vessel containing 0.06 m³ at air at 40 atmospheres and 40°C is connected with another closed vessel containing 1.35 kg of argon at 7 atmospheres. When the valve separating these vessels is opened, the pressure of the mixture is 18 atmospheres and its temperature is 30°C. Determine the volume of the vessel containing the argon and the temperature of the argon prior to mixing.
 - (b) A machine operates on an ideal cycle comprised of isentropic compression of the working substance, isometric heat addition to the working substance, isentropic expansion of the working substance and isometric heat rejection from the working substance. The working substance is assumed to be air with $\gamma=1.4$ and R=0.287 kJ/kgK. At the beginning of the compression process, the pressure is 138 kPa and the temperature is 37°C. The compression ratio is 8:1 and the maximum temperature in the cycle is 1772 K. Compute the other temperatures and pressures and indicate these values on a P-v and a T-s diagram for the cycle. What is the thermal efficiency of the cycle ? Determine the power output corresponding to a heat addition of 370 W.
- 2. (a) Steam under a pressure of 1 MPa enters a throttling calorimeter and emerges as superheated steam at atmospheric pressure with a temperature of 150°C. What is the quality of the wet steam?
 - (b) The steam enters a turbine with a velocity of 1.5 m/s and leaves at 7.5 kPa and 78.3% quality with a velocity of 90 m/s while the heat loss is 527 W and the flowrate is 0.45 kg/s. What power will be developed?
 - (c) It can easily be seen by consulting the steam tables that the process is isentropic, despite the fact that heat was lost by the turbine. Explain how this is possible.
- The diagram at the right is 3. illustrative of a gas turbine power plant being developed for use in an automobile. Assuming that the temperatures indicated have been expressed in degrees Celsius (a) draw a schematic diagram for the gas turbine power plant (b) indicate each of the processes on T-s (c) calculate the diagram efficiency for thermal operating conditions given and (d) determine the air flowrate required to develop 50 kW at the rear axle of the car.



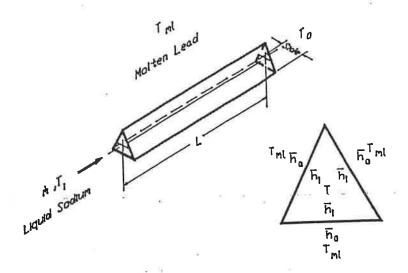


4. An ice-making refrigeration plant having a capacity of 70 kW of refrigeration operates with ammonia as the refrigerant. The evaporator operates at -15°C and the ammonia condenses at 31°C. Assuming that the compressor has an efficiency of 95% and neglecting piping losses, determine the power required per kW of refrigeration and the coefficient of performance. If cooling water enters the condenser at 20°C and leaves the condenser at 27°C, what cooling water flowrate is required?

PART B - HEAT TRANSFER

- 5. A steel tube 150 mm I.D. x 10 mm W.T. conveys wet steam at 200°C and is surrounded by air at 27°C. The loss of heat under these conditions is 2000 W per metre of length. Determine the loss of heat when the pipe is covered by a 50 mm thick layer of insulation, when the conductivities of the insulation and the pipe are respectively 0.35 W/m°C and 45 W/m°C and the heat transfer coefficient between metal and air is 40% greater than the heat transfer coefficient between insulation and air.
- 6. Heat is generated in a 75 mm diameter cylindrical electrical conductor made of a material having a thermal conductivity of 70 W/m°C which is located in a fluid having a temperature of 27°C. The surface heat transfer coefficient is 568 W/m²°C and the temperature must not exceed 540°C anywhere in the cylinder.
 - (a) Determine the maximum heat generation rate per unit length.
 - (b) Determine the surface temperature under these conditions.

A device is to be built that will 7. be capable of transferring heat from a pool of molten lead at 600 K to liquid sodium at 478 K equilateral entering an triangular duct immersed in the pool which is 4 m long by 3 cm on a side. The liquid sodium duct the flows through 3.6 kg/s and the specific heat of liquid sodium is 1340 J/kg C. Given that the heat transfer coefficient inside the duct \overline{h}_i = 89,140 W/m2°C and that the heat transfer coefficient outside the duct $\bar{h}_a = 8687 \text{ W/m}^2 ^{\circ}\text{C}$, determine



- (i) the temperature at which the liquid sodium leaves the duct
- (ii) the heat transfer between the molten lead and the liquid sodium
- 8. Water flowing at 10 kg/s through 50 brass tubes 2.3 cm ID by 2.6 cm OD heats air flowing at 1.6 kg/s through a shell within which the tubes are located in parallel. Each tube is 6.7 m long. The heat transfer coefficients inside and outside the tubes are 470 W/m²°C and 210 W/m²°C respectively. The water enters the tubes at 75°C and the air enters the shell at 15°C. Determine the heat exchanger effectiveness, the rate of heat transfer between the water and the air and the outlet temperatures of the water and the air.

Thermodynamic Properties of Ammonia

Saturated Ammonia

Temp.	Abs. Press. kPa	Specific Volume m³/kg			Enthalpy kJ/kg			Entropy kJ/kg K		
		Sat. Liquid U _f	Evap.	Sat. Vapor	Sat. Liquid	Evap.	Sat. Vapor	Sat. Liquid	Evap.	Sat. Vapor . s _o
Ü	P		75						,-	
-50.	40.88	0.001 424	2.6239	2.6254	-44.3	1416.7	1372.4	-0.1942	6.3502	6.1561
-48	45.96	0:001 429	2.3518	2.3533	-35.5	1411.3	1375.8	-0.1547	6.2696	6.1149
-46	51.55	0.001 434	2.1126	2.1140	-26.6	1405.8	1379.2	-0.1156	6.1902	6.0746
-44 .	57.69	0.001 439	1.9018	1.9032	-17.8	1400.3	1382.5	-0.0768	6.1120	6.0352
-42	64.42	0.001 444	1.7155	1.7170	-8.9	1394.7	1385.8	-0.0382	6.0349	5.9967
-40	71.77	0.001 449	1.5506	1.5521	0.0	1389.0	1389.0	0.0000	5.9589	5.9589
-38	79.80	0.001 454	1.4043	1.4058	8.9	1383.3	1392.2	0.0380	5.8840	5.9220
-36	88.54	0.001 460	1.2742	1.2757	17.8	1377.6	1395.4	0.0757	. 5.8101	5.8858
-34	98.05	0.001 465	1.1582	1.1597	26.8	1371,8	1398.5	0.1132	5.7372	5.8504
-32	108.37	0.001 470	1.0547	1.0562	35.7	1365.9	1401.6	0.1504	5.6652	5.8156
-30°	119.55	0.001 476	0.9621	0.9635	44.7	1360.0	1404.6	0.1873	5.5942	5.7815
-28	131.64	0.001 481	0.8790	0.8805	53.6	1354.0	1407.6	-0.2240	5.5241	5.7481
-26	144.70	0.001 487	0.8044	0.8059	62.6	1347.9	1410.5	0.2605	5.4548	5.7153
-24	158.78	0.001 492	0.7373	0.7388	71.6	1341.8	1413.4	0.2967	5.3864	5.6831
-24	173.93	0.001 498	0.6768	0.6783	80.7	1335.6	1416.2	0.3327	5.3188	5.6515
-22 -20	190:22	0.001 504	0.6222	0.6237	89.7	1329.3	1419.0	0.3684	5.2520	5.6205
	207.71	0.001 50 2	0.5728	0.5743	98.8	1322.9	1421.7	0.4040	5.1860	5.5900
-18 -16	226.45	0.001 515	0.5280	0.5296	··· 107.8 ···	1316.5	1424.4	0.4393	5.1207	5.5600
	246.51	0.001 513	0.4874	0.4889	116.9	1310.0	1427.0	0.4744	5.0561	5.5305
−14 −12	267.95	0.001 521	0.4505	0.4520	126.0	, 1303,5	1429.5	0.5093	4.9922	5.5015
										5.4730
-10	290.85		0.4169	0.4185	135.2	1296.8	1432.0	0.5440	4.9290	5.4449
-8	315.25	0.001 540	0.3863	0.3878	144.3	1290.1	1434.4	0.5785	4.8664	5.4173
-6	341.25	0.001 546	0.3583	0.3599	153.5	1283.3	1436.8	0.6128	4.8045	-5.3901
-4	368.90	0.001 553	0.3328	0.3343	162.7	1276.4	1439.1	0.6469	4.7432	5.3633
-2	398.27	0.001 559	0.3094		171.9	1269.4	1441.3	0.6808	4.6825	5.3369
. 0	429.44	0.001 566	0.2879	0.2895	181.1	1262.4	1443.5	0.7145	4.6223	5.3303
2	462.49	0.001 573	0.2683	0.2698	190.4	1255.2	1445.6	0.7481	4.5627	5.2852
2 4 6	497.49	0.001 580	0.2502	0.2517	199.6	1248.0	1447.6	0.7815	4.5037	
.6	534.51	0.001 587	0.2335	0.2351	208.9	1240.6	1449.6	0.8148	4.4451	5.2350
8	573.64	0.001 594	0.2182	0.2198	218.3	1233.2	1451.5	0.8479	4.3871	5.2330
10	614.95	0.001 601	0.2040	0.2056	227.6	1225.7	1453.3	0.8808		5.1861
12	658 .5 2	0.001 608	0.1910	0.1926	237.0	1218.1	1455.1	0.9136	4.2725	5.1621
14	704.44	0.001 616	0.1789	0.1805	246.4	1210.4	1456.8	0.9463	4.2159	5.1385
16	752.79	0.001 623	0.1677	0.1693	255.9	1202.6	1458.5	0.9788	4.1597	5.1151
18	803.66	0.001 631	0.1574	0.1590	265.4	1194.7	1460.0	1.0112	4.1039	5.0920
20	857.12	0.001 639	0.1477	0.1494	274.9	1186.7	1461.5	1.0434	4.0486	5.0692
22	913.27	0.001 647	0.1388	0.1405	284.4	1178.5	1462.9	1.0755	3.9937	5.0467
24	972.19	0.001 655	0.1305	0.1322	294.0	1170.3	1464.3	1.1075	3.9392	5.0244
26	1033.97	0.001 663	0.1228	0.1245	303.6	1162.0	1465.6	1.1394	3.8850	5.0023
28	1098.71	0.001 671	0.1156	0.1173	313.2	1153.6	1466.8	1.1711	3.8312	4.9805
30	1166.49	0.001 680	0.1089	0.1106	322.9	1145.0	1467.9	1.2028	3.7777	4.9589
32	1237.41	0.001 689	0.1027	0.1044	332.6	1136.4	1469.0	1.2343	3.7246	
34	1311.55	0.001 698	0.0969	0.0986	342.3	1127.6	1469.9		3.6718	4.9374
36	1389:03	0.001 707	0.0914	0.0931	3 52.1	1118.7	1470.8		3.6192	4.9161
38	1469.92	0.001 716	0.0863	0.0880	361.9	1109.7	1471.5	1.3281	3.5669	4:8950
40	1554.33	0.001 726	0.0815	0.0833	371.7	1100.5	1472.2		3.5148	4.8740
42	1642.35	0.001 735	0.0771	0.0788	381.6	1091.2	1472.8	1.390 I	3.4630	
44	1734.09	0.001 745	0.0728	0.0746	391.5	1081.7	1473.2	1.4209	3.4112	
46	1829.65	0.001 756	0.0689	0.0707	401.5	1072.0	1473.5	1.4518	3.3595	
48	1929.13	0.001 766	0.0652	0.0669	411.5	1062.2	1473.7	1.4826	3.3079	4.7905
50	2032.62	0.001 777	0.0617	0.0635	421.7	1052.0	1473.7	1.5135	3.2561	4.7696

<i>T</i> °C	u m³/kg	u kJ/kg	h kJ/kg	s kJ/kg · K	ບ m³/kg	u kJ/kg	h kJ/kg	kJ/kg · K
<u> </u>		bars = 1		-	p = 14.0			
	$p = 12.0$ $(T_{\rm int} = 3$		ZU IVLEH		$p = 14.0$ $(T_{\text{act}} = 3$	60		
	(1 m) - 5)%	(1 Mt - 2			
at.	0.10751	1337.52	1466.53	4.9625	0.09231	1339.56	1468.79	4.9050
40	0.11287	1359.73	1495.18	5.0553	0.09432	1349.29	1481.33	4.9453
60	0.12378	1404.54	1553.07	5.2347	0.10423	1396.97	1542.89	5.1360
80	0.13387	1445.91	1606.56	5.3906	0.11324	1440.06	1598.59	5.2984
00	0.14347	1485.55	1657.71	5.5315	0.12172	1480.79	1651.20	5.4433
20	0.15275	1524.41	1707.71	5.6620	0.12986	1520.41	1702.21	5.5765
40	0.16181	1563.09	1757.26	5.7850	0.13777	1559.63	1752.52	5.7013
60	0.17072	1601:95	1806.81	5.9021	0.14552	1598.92	1802.65	5.8198
80	0.17950	1641.23	1856.63	6.0145	0.15315	1638.53	1852.94	5.9333
.00	0.18819	1681.05	1906.87	6.1230	0.16068	1678.64	1903.59	6.0427
20	0.19680	1721.50	1957.66	6.2282	0.16813	1719.35	1954.73	6.148,5
40	0.20534	1762.63	2009.04	6.3303	0.17551	1760.72	2006.43	6.2513
60	0.21382	1804.48	2061.06	6.4297	0.18283	1802.78	2058.75	6.3513
80	0.22225	1847.04	2113.74	6.5267	0.19010	1845.55	2111.69	6.4488
	$T_{\rm mt} = 4$	· · · · ·			$(T_{\rm sat} = 4$	10 (16305)		
at.	0.08079	1340.97	1470.23	4.8542	0.07174	1341.88	1471.01	4.8086
60	0.08951	1389.06	1532.28	5.0461	0.07801	1380.77	1521.19	4.9627
80	0.09774	1434.02	1590.40	5.2156	0.08565	1427.79	1581.97	5.1399
00	0.10539	1475.93	1644.56	5.3648	0.09267	1470.97	1637.78	5.2937
20	0.11268	1516.34	1696.64	5.5008	0.09931	1512.22	1690.98	5,4326
40	0.11974	1556.14	1747.72	5.6276	0.10570	1552.61	1742.88	5.5614
60	0.12663	1595.85	1798.45	5.7475	0.11192	1592.76	1794.23	5.6828
.80	0.13339	1635.81	1849.23	5.8621	0.11801	1633.08	1845.50	5.7985
00	0.14005	1676.21	1900.29	5.9723	0.12400	1673.78	1896.98	5.9096
20	0.14663	1717.18	1951.79	6.0789	0.12991	1715.00	1948.83	6.0170
40	0:15314	1758.79	2003.81	6.1823	0.13574	1756.85	2001.18	6.1210
60	0.15959	1801.07	2056.42	6.2829	0.14152	1799.35	2054.08	6.2222
80	0.16599	1844.05	2109.64	6.3809	0.14724	1 1842.55	1 2107.58	6.3207
	(0)						554	**
	$p = 20.0$ $(T_{\rm mi} = 4$	bars = 2	.00 MPa					.
at.	0.06445	1342.37	1471.26	4.7670				
60	0.06875	1372.05	1509.54	4.8838				82
80	0.07596	1421.36	1573.27	5.0696				
00	0.08248	1465.89	1630.86	5.2283				
20	0.08861	1.508.03	-1685.24	5.3703			*	H X (2)
40	0.09447	1549.03	1737.98	5.5012	*		10	
ś	0.10016	1500 65	-1789.97-	5 6241-	* 100			

5.6241⁻ 5.7409

5.8530

5.9611

6.0658 6.1675

0.10016 1589.65 1789.97 0.10571 1630.32 1841.74 0.11116 1671.33 1893.64

1712.82

1754:90

280 0.13224 1841.03 2105.50 6.2665

4

1945.87 1998.54 2051.74

160

180

200

220

240

260

0.10571 0.11116

0.11652

0.12182

0.12706 [797.63]